## **Long Train Coupler Forces**

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HEAVY HAUL SEMINAR \* MAY 2-3, 2018









## **Presentation Outline**

- 100+ car coil steel trains
  - 80-car practical limit
  - Alternative powering possibilities
- Mixed train Multilevels and Doublestacks
  - The speed wave, and
  - How to minimize it





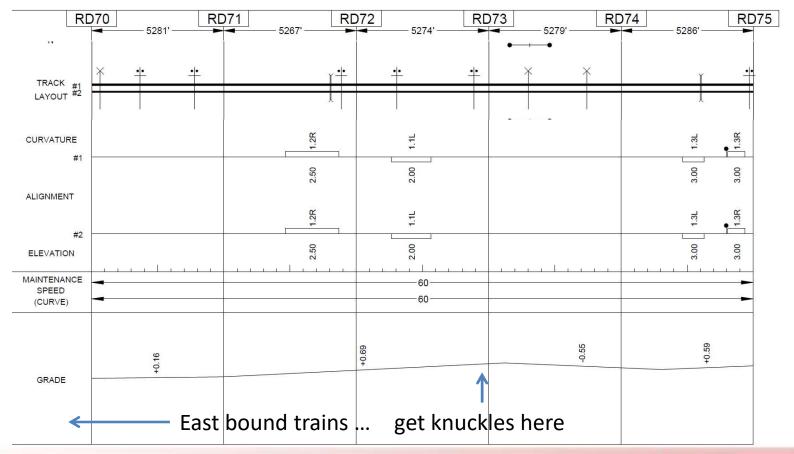


## **Coil Steel Train 60W**

- Conventional Consist: three (3) head-end units
- 100 loads; 0 mtys; 12-13,000 tons; 5500-5700 ft
- Undulating grade in east central Ohio (Alliance)
- Frequent broken knuckles near the rear of the train
- Limit train to 80 cars → no knuckles
- Why? Is there a better solution?

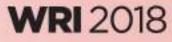












## **Consist: EOCC-equipped trough cars**

263k, 58'8"

Three head-end units

• 103 lds, 0 mtys, 12500 tons, 5500 ft

286k, 48'8"

15" EOCC units

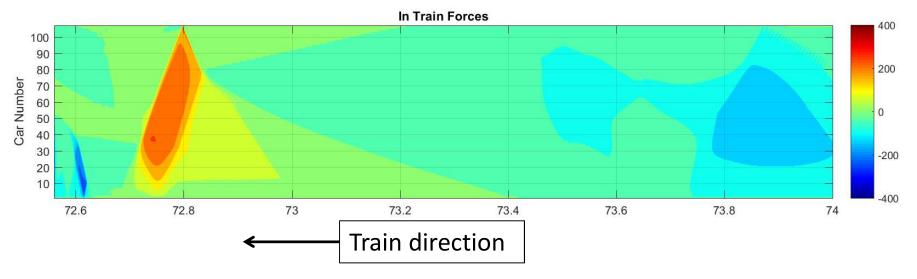
100 cars yields 250-275 ft of slack (>5 carlengths)







#### **TOES** simulation: 103-car train

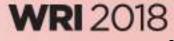


- X axis head-end mile post
- Y axis position in train (left) and coupler force (color)

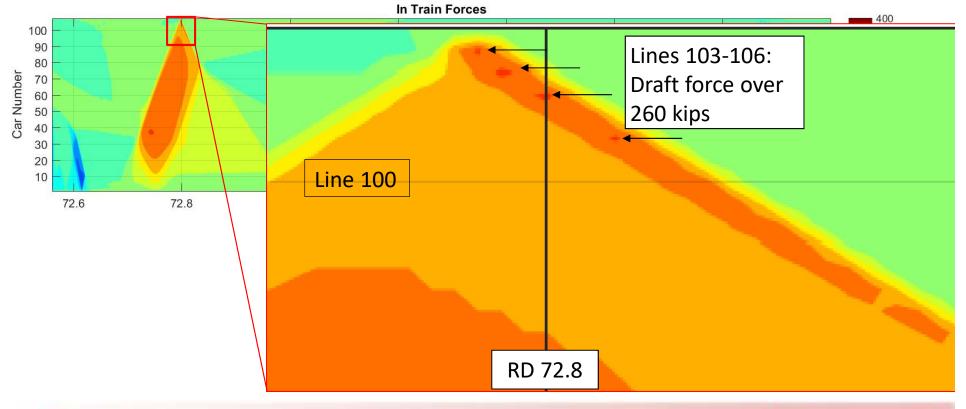
To read graph: Select head-end mile post, then read vertically to determine coupler forces throughout train at that location.







#### **TOES** simulation: 103-car train

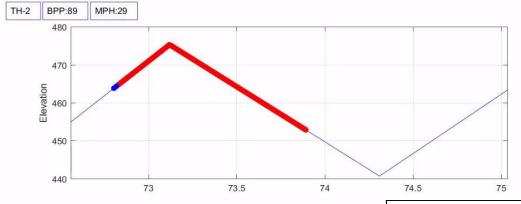




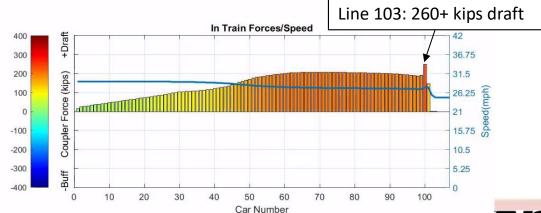


**WRI** 2018

#### **TOES** simulation: 103-car train

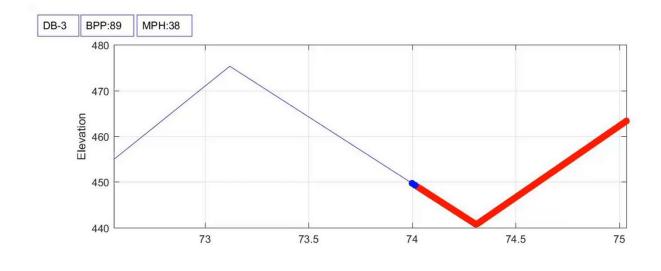


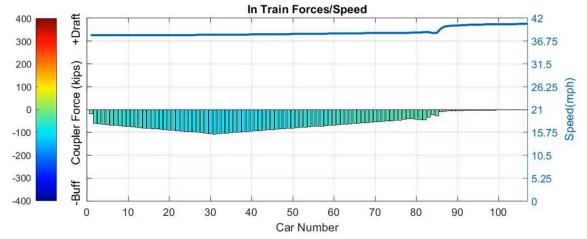
We found peak draft forces occurred suddenly at the rear of the train: "Cracking the whip."











#### 103-car train

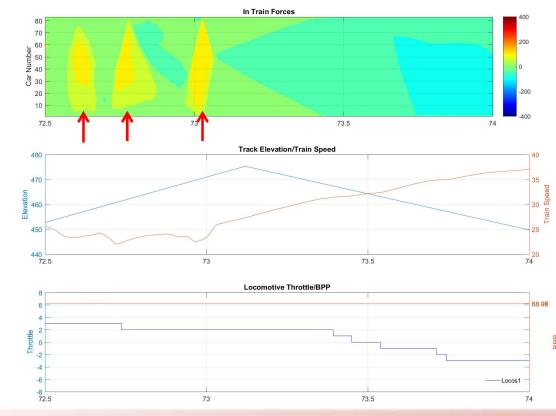




#### **TOES** simulation: 80-car train

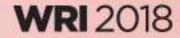
Three moderate, but abrupt, run-out events

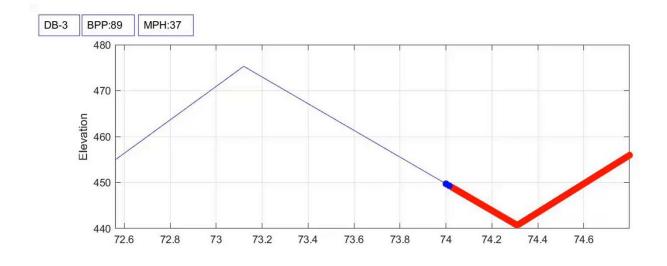
Throttle handling similar to 100-car case

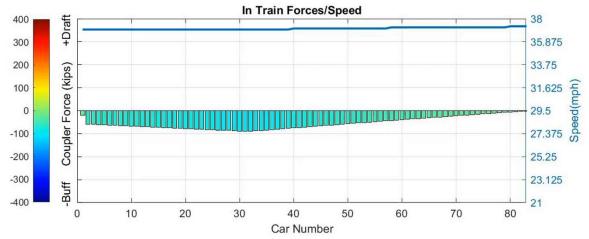












#### 80-car train



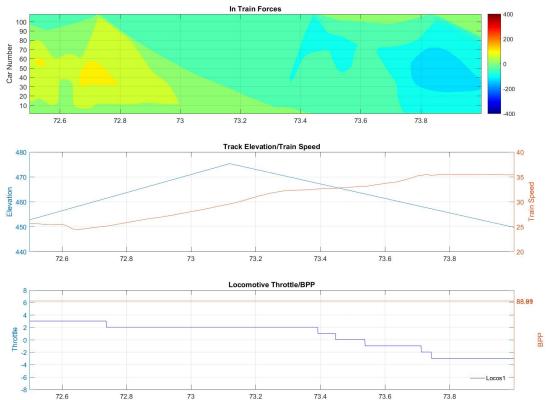
**WRI** 2018

### **TOES** simulation: DP synchronous mode

104 loads,13180 tons, 5700 ft

Note smooth transition from light buff to light draft

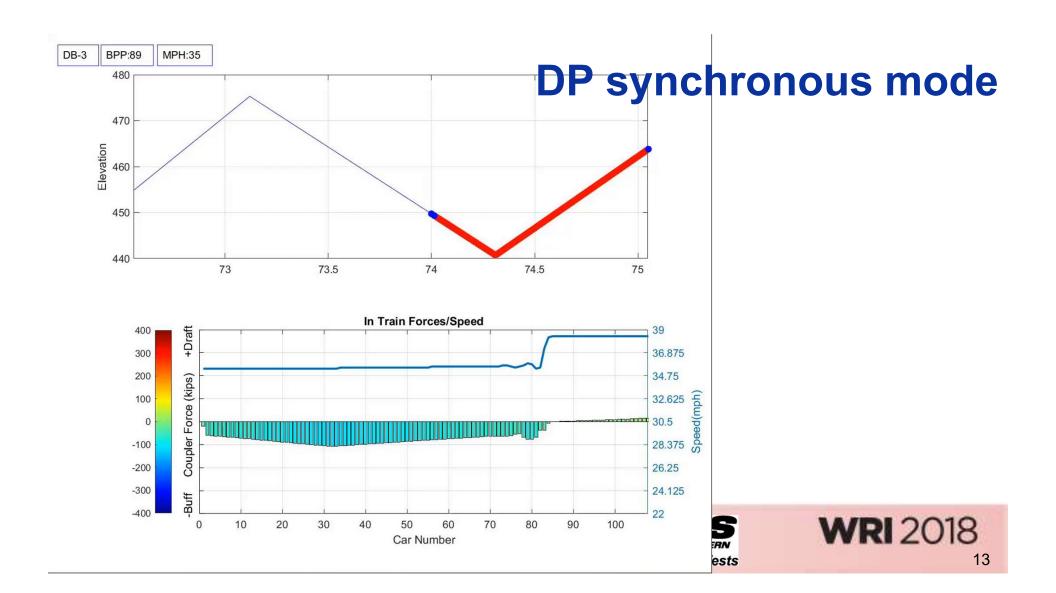
Same throttle handling as conventional case











## **60W Conclusions**

- We verified the draft forces breaking the knuckles.
- We verified 80-car trains avoid getting knuckles.
- We showed that we could use DP to manage coupler forces, and still run full-sized trains.



## 23G Derailment at Coster



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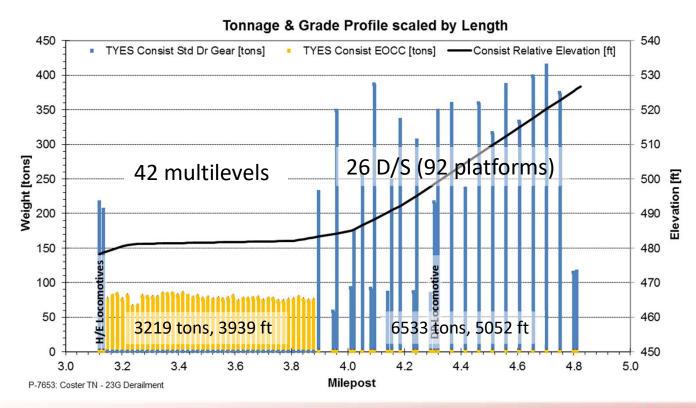
## 23G Derailment at Coster

- 2x1 mid-train DP
- 68 lds, 0 mtys, 9752 tons, 9211 ft
- 42 multilevels leading 26 doublestacks
- Undulating grade in east Tennessee
- Gage ruptured under last M/L and first D/S





#### **23G Coster Derailment**

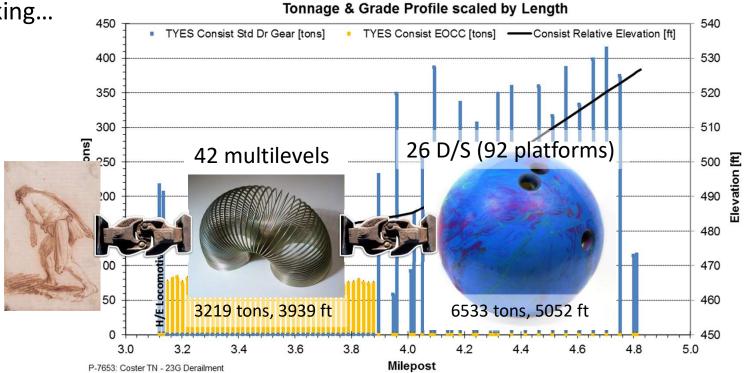






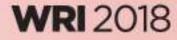
#### **23G Coster Derailment**

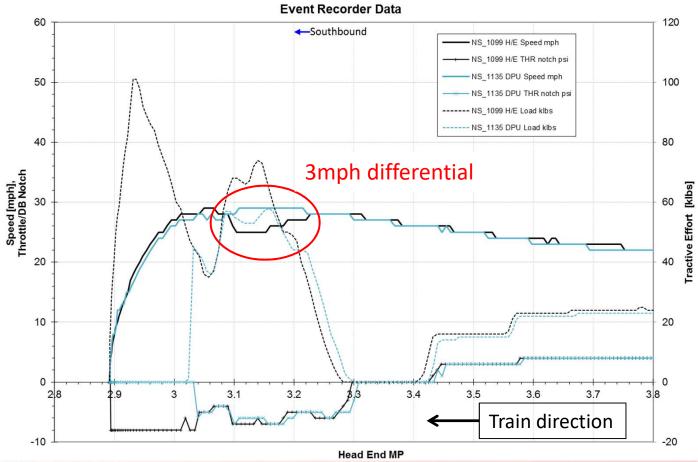
Metaphorically speaking...











P-7653: Coster TN - 23G Derailment

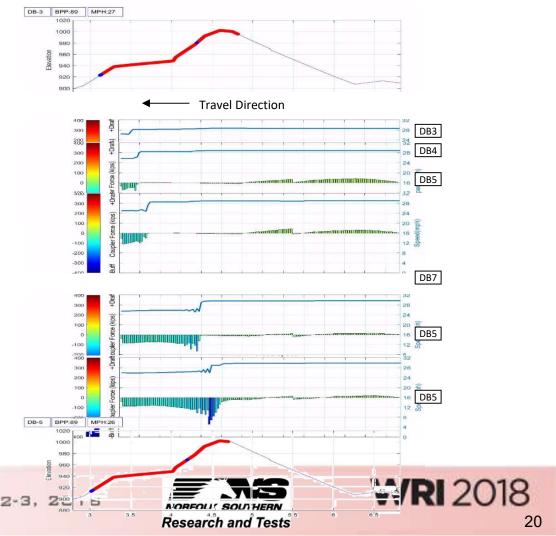
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# **TOES** simulation actual conditions

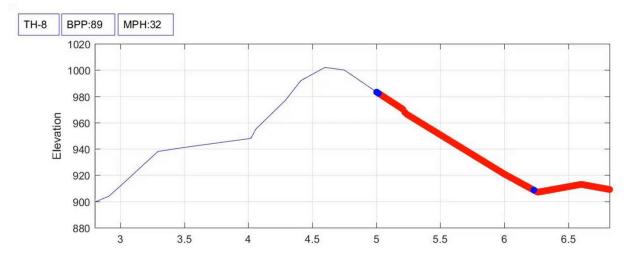
- 2x1 mid-train DP synchronous mode
- Head end decelerated 3 mph, DP accelerated 1 mph until run-in occurred

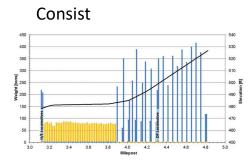




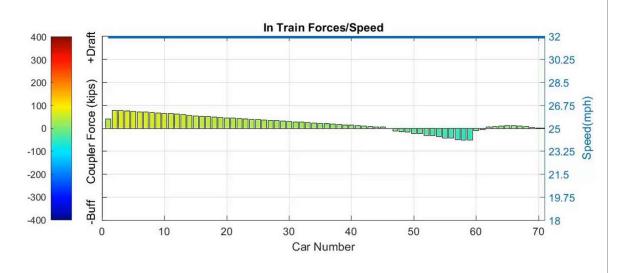
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#### **Actual conditions**



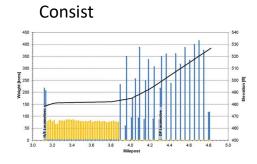




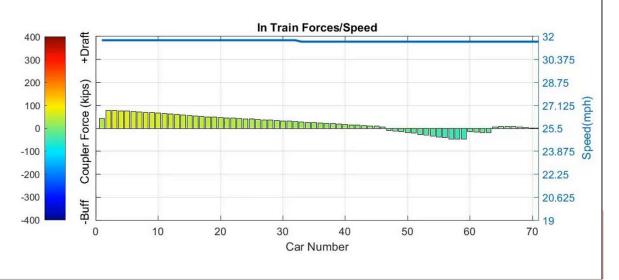


# Actual consist, fenced DP





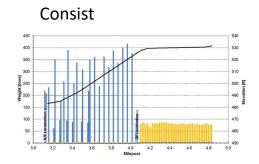




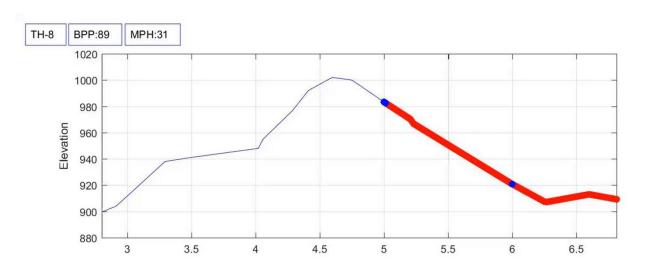
# Improved train makeup

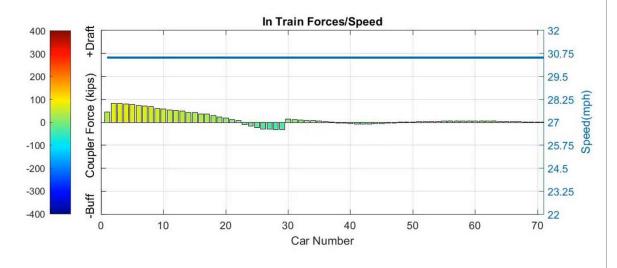
Synchronous mode

Train handling similar to original case







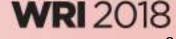


## 23G Conclusions

- Significant velocity differentials can exist in large blocks of EOCC cars.
- Very large coupler forces can arise at the coupling between EOCC and conventional cars.
- DP can help, but only so much.
- Address the root problem: EOCC block position.







## **Conclusions**

- Buff forces are more likely to result in track damage and derailment than draft forces.
- Large coupler forces tend to result from type of equipment (EOCC blocks) and tonnage (not necessarily length).



## **Questions, Comments, Discussion**





